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A DEFLECTION FORMULA FOR SINGLE-SPAN BEAMS OF CONSTANT SECTION SUBJECTED TO COMBINED AXIAL AND TRANSVERSE LOADS

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#### SUMMARY

In this paper there is presented a deflection formula for single-span beams of constant section subjected to combined axial and transverse loads of the types commonly encountered in airplane design. The form of the equation is obtainable by dimensional analysis. Tables and curves of the nondimensional coefficients are appended to facilitate the use of the formula.

The equation is applied to the determination of the spring constant of a beam. Tables and curves are presented to show the variation of the spring constant with changes in the axial load and position along the beam.

#### INTRODUCTION

In reference 1 deflection formulas are presented for single-span beams of constant section subjected to axial compression and transverse loading. These formulas are considerably different for the various loading conditions treated and must be altered when the axial load is tension.

The purpose of this report is to present a simple formula that includes all the above-mentioned cases of reference 1 and is valid when the axial load is either tension or compression.

In reference 2 is presented a detailed study of the interaction between a lift strut and a wing spar when connected by a jury strut. Therein the authors state. "It would be very interesting to make a general study of the effect of varying the axial load upon the sign and magni-

tude of the spring constant . . . . " In reference 2 the term "jury strut" is applied to a member whose primary function is to provide an elastic support to a lift strut at some intermediate point and thereby to increase the critical load of the lift strut.

The deflection formula derived in this report makes it possible to make the suggested study and to show very clearly the effect of changes in the axial load. The general deflection formula presented herein is derived by the strain-energy method of analysis which is believed to lead to a more simple form of solution. A comprehensive treatment of this method has been presented by Timoshenko. (See reference 3.)

## DERIVATION OF EQUATION FOR BEAM DEFLECTION

The following derivation applies to the case of a single-span beam of constant section subjected to the combination of loadings shown in figure 1. The loads, moments, and deflections are shown in the positive directions. The conventions adopted here are such that positive lateral loads and positive end moments produce positive deflections; also positive axial load increases deflections. It should be noted that these conventions differ from those of reference 1.

The deflection curve of the beam may be obtained by the addition of simple curves of sinusoidal form having different amplitudes and frequencies so that

$$y = a_1 \sin \frac{\pi x}{l} + a_2 \sin \frac{2\pi x}{l} + \dots + a_n \sin \frac{n\pi x}{l}$$
 (1)

In order that this expression may exactly represent a particular deflection curve at every point of the beam, an infinite number of terms are, in general, required. Therefore the expression

$$y = \sum_{n=1}^{\infty} a_n \sin \frac{n\pi x}{l}$$
 (2)

may be made to represent any deflection curve for a single-span beam by adjusting the values of the coefficients  $a_n$ . In order to evaluate the coefficients  $a_n$ , the changes of energy of the external loads and of the beam

due to bending are determined for a small change dan in any one of the coefficients  $a_n$ . For equilibrium, the changes in energy of the external loads must be equal and opposite to the change in internal energy of the beam. It is therefore necessary to find the energy changes during a small displacement dan  $\sin \frac{n\pi x}{l}$  from the equilibrium position. During this small displacement the increase in energy of the beam due to bending is  $\frac{EI\pi^4}{2l^3}$  n<sup>4</sup> and dan (reference 3, pp. 417-422).

The work done by the axial force P is

$$\vec{P} \frac{\pi^2}{2l} n^2 a_n da_n$$

The work done by the load W is

The intensity of the trapezoidal loading is

where c is the change in the intensity of loading per unit length of span. The work done by that loading is

$$\int_{0}^{l} (w_{0} + cx) da_{n} \sin \frac{n\pi x}{l} dx$$

$$= \left[ \frac{w_{0}l}{n\pi} - \frac{w_{0}l}{n\pi} (-1)^{n} - \frac{cl^{s}}{n\pi} (-1)^{n} \right] da_{n}$$

The rotations of the ends of the beam are

$$\begin{bmatrix} \frac{d \left( \operatorname{da_n} \sin \frac{n\pi x}{l} \right)}{\operatorname{dx}} \end{bmatrix}_{\mathbf{x}=0} = \frac{n\pi}{l} \operatorname{da_n} \qquad \text{at } \mathbf{x}=0$$
and
$$\begin{bmatrix} \frac{d \left( \operatorname{da_n} \sin \frac{n\pi x}{l} \right)}{\operatorname{dx}} \end{bmatrix}_{\mathbf{x}=l} = \frac{n\pi}{l} \left( -1 \right)^n \operatorname{da_n} \qquad \text{at } \mathbf{x}=l$$

During these rotations the work done by the end moments

M1 and M2 is

$$\left[\mathbf{M}_{1} \ \frac{\mathbf{n}_{\Pi}}{l} - \mathbf{M}_{3} \ \frac{\mathbf{n}_{\Pi}}{l} \ (-1)^{\mathbf{n}}\right] d\mathbf{a}_{\mathbf{n}}$$

When the work done by the external forces is equated to the change in bending energy of the beam, the following equation for  $a_n$  is obtained.

$$a_n = \frac{2l}{\pi^2 \overline{P}} \frac{\overline{\alpha}}{n^2 (n^2 - \overline{\alpha})} \left[ \left( \frac{\underline{M_1}}{l} \right) (n\pi) - \frac{\underline{M_2}}{l} (n\pi \cos n\pi) + \right]$$

$$(w_0 l) \left(\frac{1}{n\pi} - \frac{\cos n\pi}{n\pi}\right) - (c l^2) \left(\frac{\cos n\pi}{n\pi}\right) + \pi \sin \frac{n\pi d}{l}$$
 (3)

$$\overline{\alpha} = \frac{\overline{P}}{\left(\frac{\pi^2 \, \mathbb{E} \, \mathbb{I}}{l^2}\right)} = \frac{\overline{P}}{P_e} \tag{4}$$

and  $P_0$  is the critical Euler load for a pin-ended column of length l and bending rigidity EI. The axial load, P, is positive when it is compression and negative when it is tension. By definition,  $\alpha$  has the same sign as P. When the foregoing value of  $a_n$  is substituted in equation (2), the expression for the deflection curve becomes

$$y = \frac{1}{\overline{P}} \left[ \overline{\beta} \left( \frac{M_1}{l} \right) + \overline{\gamma} \left( \frac{M_2}{l} \right) + \overline{\delta} \left( w_0 l \right) + \overline{\epsilon} \left( c l^2 \right) + \left( \overline{\phi}_1 + \overline{\phi}_2 \right) W \right]$$
 (5)

The coefficients  $\overline{\beta}$ ,  $\overline{\gamma}$ ,  $\overline{\delta}$ ,  $\overline{\epsilon}$ ,  $\overline{\phi_1}$ , and  $\overline{\phi_2}$  vary in form with changes in the sign of  $\overline{\alpha}$ . It is more convenient if these coefficients are replaced by the coefficients  $\beta$ ,  $\gamma$ ,  $\delta$ ,  $\epsilon$ ,  $\phi_1$ , and  $\phi_2$  in which

$$\frac{\beta}{P} = \frac{\overline{\beta}}{\overline{P}}, \quad \frac{\gamma}{P} = \frac{\overline{\gamma}}{\overline{P}}, \quad \text{etc.}$$

where P is the absolute value of  $\overline{P}$  and  $\overline{\alpha}$  is replaced by its absolute value  $\alpha$ . In order to differentiate between compression, tension, and no axial load the coefficients  $\beta$ ,  $\gamma$ , etc., are given the subscripts  $\alpha$ , and  $\alpha$ . Equation (5) may then be written  $\alpha$ s

$$y = \frac{1}{P} \left[ \beta_{()} \left( \frac{M_1}{l} \right) + \gamma_{()} \left( \frac{M_2}{l} \right) + \delta_{()} \left( w_0 l \right) + \epsilon_{()} \left( cl^2 \right) + \left( \phi_{1()} + \phi_{2()} \right) \right]$$

$$(5a)$$

When the axial load is compression or tension, use the coefficients with subscripts c and t, respectively. When there is no axial load replace P by Pe and use the coefficients with subscript o. For axial compression, denoted by subscript c,

$$\beta_{\rm c} = \frac{2\alpha}{\pi} \sum_{\rm n=1}^{\infty} \frac{\sin \frac{n_{\rm H}x}{l}}{n({\rm r}^2-\alpha)} = \left\{ \frac{\sin \left[\sqrt{\alpha_{\rm H}}\left(1-\frac{x}{l}\right)\right]}{\sin (\sqrt{a_{\rm H}})} - \left(1-\frac{x}{l}\right) \right\}$$

$$\gamma_{c} = \frac{-3\alpha}{\pi} \sum_{n=1}^{\infty} \frac{\cos(n\pi) \sin \frac{n\pi x}{l}}{n(n^{2} - \alpha)} = -\left[\frac{x}{l} - \frac{\sin(\sqrt{\alpha\pi} \frac{x}{l})}{\sin(\sqrt{a\pi})}\right]$$

$$\delta_{c} = \frac{2\alpha}{\pi^{3}} \sum_{n=1}^{\infty} \frac{(1 - \cos n\pi) \sin \frac{n\pi x}{l}}{n^{3} (n^{2} - \alpha)} =$$

$$\left\{ \left(\frac{x}{3l}\right) \left(\frac{x}{l}-1\right) + \frac{2 \sin \left(\sqrt{\alpha} \frac{\pi}{2} \frac{x}{l}\right) \sin \left[\sqrt{\alpha} \frac{\pi}{2} \left(1-\frac{x}{l}\right)\right]}{\pi^2 \alpha \cos \left(\sqrt{\alpha} \frac{\pi}{2}\right)} \right\}$$

$$\epsilon_{o} = -\frac{2\alpha}{\pi^{3}} \sum_{n=1}^{\infty} \frac{\cos n\pi}{n^{3} (n^{2} - \alpha)} = -\left\{ \frac{1}{\pi^{3} \alpha} \left[ \frac{x}{t} - \frac{\sin(\sqrt{\alpha}\pi)}{\sin(\sqrt{\alpha}\pi)} \right] + \frac{1}{\delta\alpha} \left( \frac{x}{t} - \frac{x^{3}}{t^{3}} \right) \right\}$$

(Dr. Kaplan of this Laboratory assisted in the evaluation of the foregoing infinite series)

and 
$$\phi_{1_{C}} + \phi_{2_{C}} = \frac{2\alpha}{\pi^{2}} \sum_{n=1}^{\infty} \frac{\sin \frac{n\pi x}{l} \sin \frac{n\pi d}{l}}{n^{2} (n^{2} - \alpha)}$$
where 
$$\phi_{1_{C}} = \left\{ \frac{\underline{u}}{2} \left( 1 - \frac{\underline{u}}{2} \right) - \frac{\cos \left[ \sqrt{\alpha}\pi (\underline{u} - 1) \right]}{2\sqrt{\alpha}\pi \sin(\sqrt{\alpha}\pi)} \right\}$$
where 
$$\underline{u} = \left( \frac{\underline{d}}{l} - \frac{\underline{x}}{l} \right)$$
and 
$$\phi_{2_{C}} = -\left\{ \frac{\underline{v}}{2} \left( 1 - \frac{\underline{v}}{2} \right) - \frac{\cos \left[ \sqrt{\alpha}\pi (\underline{v} - 1) \right]}{2\sqrt{\alpha}\pi \sin(\sqrt{\alpha}\pi)} \right\}$$
where 
$$\underline{v} = \left( \frac{\underline{d}}{l} + \frac{\underline{x}}{l} \right)$$

For axial tension, denoted by subscript t,  $\overline{\alpha}$  is negative and the coefficients take the following forms.

$$\beta_{t} = -\left\{\frac{\sinh\left[\sqrt{\alpha_{H}}\left(1-\frac{x}{l}\right)\right]}{\sinh\left(\sqrt{\alpha_{H}}\right)} - \left(1-\frac{x}{l}\right)\right\}$$

$$\gamma_{t} = \begin{bmatrix} \frac{x}{l} - \frac{\sinh\left(\sqrt{\alpha_{H}}\frac{x}{l}\right)}{\sinh\left(\sqrt{\alpha_{H}}\right)} \end{bmatrix}$$

$$\delta_{t} = \left\{\left(\frac{x}{2l}\right)\left(\frac{x}{l} - 1\right) - \frac{2\sinh\left(\sqrt{\alpha_{H}}\frac{x}{l}\right)}{\sinh\left(\sqrt{\alpha_{H}}\frac{x}{l}\right)}\right\}$$

$$\epsilon_{t} = \left\{\frac{1}{\pi^{3}\alpha}\left[\frac{x}{l} - \frac{\sinh\left(\sqrt{\alpha_{H}}\frac{x}{l}\right)}{\sinh\left(\sqrt{\alpha_{H}}\right)}\right] - \frac{1}{6\alpha}\left(\frac{x}{l} - \frac{x^{3}}{l^{3}}\right)\right\}$$

$$\phi_{1} = \left\{\frac{u}{l}\left(1-\frac{u}{l}\right) + \frac{\cosh\left[\sqrt{\alpha_{H}}\left(u-1\right)\right]}{2\sqrt{\alpha_{H}}\sinh\left(\sqrt{\alpha_{H}}\right)}\right\}$$
where 
$$v = \left(\frac{d}{l} - \frac{x}{l}\right)$$
where 
$$v = \left(\frac{d}{l} + \frac{x}{l}\right)$$

When there is no axial load, replace P by  $P_{\rm e}$ , in equation (5a), and use the coefficients with subscript zero.

$$\beta_{0} = \frac{\Pi^{2}}{6} \frac{\mathbf{x}}{l} \left( 1 - \frac{\mathbf{x}}{l} \right) \left( 2 - \frac{\mathbf{x}}{l} \right)$$

$$\gamma_{0} = \frac{\Pi^{2}}{6} \frac{\mathbf{x}}{l} \left\{ 1 - \left( \frac{\mathbf{x}}{l} \right)^{3} \right\}$$

$$\delta_{0} = \frac{\Pi^{2}}{24} \left\{ \frac{\mathbf{x}}{l} - 2 \left( \frac{\mathbf{x}}{l} \right)^{3} + \left( \frac{\mathbf{x}}{l} \right)^{4} \right\}$$

$$\epsilon_{0} = \frac{\Pi^{2}}{360} \left\{ 7 \left( \frac{\mathbf{x}}{l} \right) - 10 \left( \frac{\mathbf{x}}{l} \right)^{3} + 3 \left( \frac{\mathbf{x}}{l} \right)^{5} \right\}$$

$$\phi_{1_{0}} = \pi^{3} \left\{ \frac{1}{90} - \frac{u^{3}}{12} + \frac{u^{3}}{12} - \frac{u^{4}}{48} \right\}$$
where
$$u = \left( \frac{d}{l} - \frac{x}{l} \right)$$

$$\phi_{2_{0}} = -\pi^{2} \left\{ \frac{1}{90} - \frac{v^{2}}{12} + \frac{v^{3}}{12} - \frac{v^{4}}{48} \right\}$$
where
$$v = \left( \frac{d}{l} + \frac{x}{l} \right)$$

The foregoing coefficients are given in tables I to IV and figures 5 to 16 for the range likely to be encountered in airplane design.

When more than one concentrated load acts on the beam, the deflection formula becomes

$$y = \frac{1}{P} \left[ \beta_{()} \left( \frac{M_1}{l} \right) + \gamma_{()} \left( \frac{M_2}{l} \right) + \delta_{()} \left( w_0 l \right) + \epsilon_{()} \left( c l^2 \right) + \Sigma \left( \phi_{1()} + \phi_{2()} \right) \right]$$

$$(6)$$

where the summation sign indicates that there must be one term of that form for each concentrated load. When the values of  $\varphi_1$  and  $\varphi_2$  are being chosen, the following rule must be observed. If the point whose deflection is being determined lies to the left of the concentrated load being considered, use the values of  $\frac{X}{l}$  and  $\frac{d}{l}$  as in the derivation. When the deflection is being determined for a point to the right of the load, however, replace  $\frac{X}{l}$  and  $\frac{d}{l}$  by  $\left(1-\frac{X}{l}\right)$  and  $\left(1-\frac{d}{l}\right)$ , respectively, in computing u and v.

An inspection of the curves and tables shows that when the axial load is compression the deflections become excessive as a approaches unity, even for small lateral loads. A comprehensive treatment of critical loading conditions is given in reference 2.

# DETERMINATION OF SPRING CONSTANT

The spring constant of a beam at any point is defined as the lateral force required at that point to produce unit deflection of that point. The spring constant K may be defined (fig. 2) mathematically as

$$\mathbf{K} = \left[ \frac{9}{9} \mathbf{M} \right]^{\mathbf{x} = \mathbf{q}} \tag{7}$$

Equation (5a) may be applied to figure 2 and is reducible to, the form

$$y = \frac{\pi l}{P_{\theta}} \zeta \tag{8}$$

For axial compression

$$\zeta_{c} = -\frac{1}{\alpha} \left( \frac{\mathbf{x}}{l} \right) \left( 1 - \frac{\mathbf{x}}{l} \right)$$

$$+ \frac{\sqrt{\alpha_{H}} \left\{ \cos \left[ \sqrt{\alpha_{H}} \left( \frac{3\mathbf{x}}{l} - 1 \right) \right] - \cos \left( \sqrt{\alpha_{H}} \right) \right\}}{3 \alpha^{2} \pi^{2} \sin \left( \sqrt{\alpha_{H}} \right)}$$

and for axial tension

$$\xi_{t} = \frac{1}{\alpha} \left( \frac{\mathbf{x}}{l} \right) \left( 1 - \frac{\mathbf{x}}{l} \right) + \frac{\sqrt{\alpha}\pi \left\{ \cosh \left[ \sqrt{\alpha}\pi \left( \frac{2\mathbf{x}}{l} - 1 \right) \right] - \cosh \left( \sqrt{\alpha}\pi \right) \right\}}{2 \alpha^{2} \pi^{2} \sinh \left( \sqrt{\alpha}\pi \right)}$$

When there is no axial load present

$$\zeta_0 = \frac{\pi^2}{3} \frac{x^2}{x^3} \left( \frac{1}{x} - 1 \right)^3$$

Therefore, the spring constant K may be written as

$$\mathbf{K} = \mathbf{K}_1 \frac{\mathbf{P}_{\mathbf{S}}}{l} \tag{9}$$

where

$$K_1 = \frac{1}{\zeta}$$

or, in the alternative form, which may be more convenient at times

$$K = K_2 \frac{P}{l} \tag{10}$$

where

$$K = K_a \frac{P}{l}$$

$$K_a = \frac{1}{\alpha l}$$

The values of  $\zeta$ ,  $K_1$ , and  $K_2$  are presented in tables V to VIII and in figures 17 to 24.

The effect of changes in axial load upon the sign and magnitude of the spring constant may be seen by an inspection of curves of  $K_1$  against  $\alpha$  for various positions along the span.

### PRACTICAL APPLICATIONS

Equation (6) may readily be used in preliminary or final design to compute the deflections of beams of constant section subjected to combined axial and transverse loads. Its form is simple and the tables and charts reduce the amount of computation to a minimum. It may also be used to compute the additional deflections and from them the additional bending moments due to the load in a jury strut.

When the deflection of any point on a beam is known, the bending moment at that point may be written as

$$M = M_0 \pm Py$$

where  $M_0$  is the bending moment at that point neglecting the effect of beam deflection and  $\pm Py$  is the bending moment due to the axial load P and the beam deflection y at the point. Equation (6) may be used to determine the value of y for the cases treated in this note.

Equation (10) has a special significance in jury-strut problems. In reference 2, equations are given for the required minimum value of the spring constant of a lift strut for various conditions. A lift strut will have minimum weight when its value of  $\alpha$  is so chosen that it just develops the required spring constant. The tables and

curves computed from equation (10) enable the designer to select the lightest strut for any particular case.

Langley Memorial Aeronautical Laboratory,
National Advisory Committee for Aeronautics,
Langley Field, Va., July 15, 1935.

## APPENDIX A

In order to demonstrate the application of the general equation to a specific case, the following problem is considered. Referring to figure 3, let it be required to determine (a) the deflection of point A due to the external loads, exclusive of the reaction of the jury strut (AB), (b) a suitable lift strut, and (c), the supporting force on the spar.

		Spar	<u>s</u> ,t	rut	
'E'	= .	+33,300 in1b.		0	
H <sub>2</sub>	=	0 .		0	
P	=	5170 lb. tension	5,450 1	Lb.	compression
I	=	150 in.4			, -
$\left(\frac{\mathbf{x}}{l}\right)_{\mathbf{A}}$	=	C.30		0.30	0
E	=	1,300,000 lb./sq.in.	28,000,	000	lb./sq.in.
P <sub>e</sub>	== '	$\frac{9.88 \times 1300000 \times 150}{(150)^2} = 85600$	0 1b.		
α	<del></del>	$\frac{P}{P_e} = \frac{5170}{85600} = 0.0604$			
Ψo	= •	-10 lb./in.			

(a) The deflection at point A, due to the external loads exclusive of the jury-strut reaction, is given by

$$y_{A} = \frac{1}{F} \left[ \beta_{t} \frac{M_{1}}{l} + \delta_{t} \left( w_{0} l \right) \right]$$

$$= \frac{150}{5170} \left[ (0.0335) \left( \frac{+33300}{150} \right) + (0.0059) \left( -10 \times 150 \right) \right]$$

$$= -0.041 \text{ in.}$$

 $eta_{t}$  from figure 7;  $\delta_{t}$  from figure 10

(b) From figure 3

$$\cos^2 \mu = \left(\frac{150}{158.11}\right)^2 = 0.9$$

The spring constant of the spar at point A is (equation (9))

$$K_{1_{t}} \frac{P_{\Theta}}{l} = 7.2 \frac{85600}{150} = +4110$$
 $(K_{1_{t}} \text{ from fig. 22})$ 

In the notation of this paper the maximum allowable negative value of the spring constant of the lift strut is (reference 2, equation (22)),

$$-K_{(spar)} \stackrel{cos^2}{\cdot} \mu = K_{(strut)}$$
  
 $K_{(strut)} = -(4,110) (0.9) = -3,700$ 

and from equation (10)

$$(K_2)_{\text{strut}} = \frac{-3700 \times 158.11}{5450} = -107.3$$

From figure 23 it may be seen that for  $\frac{X}{l}=0.3$  the maximum allowable value of  $\alpha$  for the strut lies between  $\alpha=3.18$  and  $\alpha=3.20$ . If a strut be chosen with this value, it will be in a condition of indifferent elastic stability. For positive stability the strut chosen should have a value of  $K_{2c}$  algebraically greater than -107.3. The optimum strut will, in general, have a value of  $K_{2c}$  given by the point on its curve at which the slope begins to change rapidly. (See reference 2.) In this example the

optimum strut is at  $\alpha=2.8$ , approximately, at which  $K_2=\sim10.5$ . When this value of  $\alpha=2.8$  has been chosen, the moment of inertia of the strut is obtained from

$$I = \frac{l^2}{\pi^2 E} \frac{P}{\alpha} = \frac{(158.11)^2}{(9.88)(28000000)} \times \frac{5450}{2.8} = 0.176 \text{ in.}^4$$

(c) The corrected value of the spring constant of this strut is (reference 2, equation (22))

$$K = \frac{5450}{158.11} \times \frac{1}{0.9} (-10.5) = -402$$

In the notation of this paper the supporting load on the spar is (reference 2, equation (9))

$$W_0 = -\frac{X_{\text{(spar)}} X_{\text{(strut)}} Y_A}{X_{\text{(spar)}} + X_{\text{(strut)}}}$$

$$W_0 = -(-0.041) \frac{4110(-402)}{4110-402} = -18.3 \text{ lb.}$$

The minus sign in the foregoing equation indicates that the supporting load acts in the same direction as the lateral load.

## APPENDIX B.

### Beams with Restrained Ends

In the derivation of equation (6) the values of  $k_1$  and  $k_2$  were assumed to be known. It is possible, however, to apply the general equation to the solution of problems in which the end moments must first be determined.

Let it be required to determine the moments  $\aleph_1$  and  $\aleph_2$  for the problem in figure 4.

The deflection at any point is

$$y = \frac{1}{P} \left[ -\frac{w_1}{l} \beta_c - \frac{w_2}{l} \gamma_c + \delta_c (w_0 l) \right]$$

The slope at any point is

$$\frac{dy}{dx} = \frac{l}{P} \left[ -\frac{M_1}{l} \frac{d\beta_C}{dx} - \frac{M_2}{l} \frac{d\gamma_C}{dx} + (w_0 l) \frac{d\delta_C}{dx} \right]$$

The end moments are equal due to symmetry and, since the slopes at the ends are zero.

$$\mathbf{H}_{1} = \mathbf{H}_{2} = \mathbf{H} = \mathbf{W}_{0} l^{2} \frac{\frac{d\delta_{C}}{dx}}{\frac{d\beta_{C}}{dx} + \frac{d\gamma_{C}}{dx}}$$

A close approximation may be obtained by substituting

 $\frac{\Delta \beta}{\Delta x}$ ,  $\frac{\Delta \gamma}{\Delta x}$ , and  $\frac{\Delta \delta}{\Delta x}$  for  $\frac{d\beta}{dx}$ ,  $\frac{d\gamma}{dx}$ , and  $\frac{d\delta}{dx}$ , respectively, the values of the former being readily obtainable from tables I and II. When the values thus obtained are substituted in the preceding equation,

$$M = w_0 l^2 \frac{\frac{0.0203}{0.05}}{\frac{0.1282}{0.05} + \frac{0.0893}{0.05}} = 0.0934 w_0 l^2$$

In reference 4, equation (6.128), the exact solution for this case yields the result

$$H = 0.091 \text{ wol}^2$$

More complicated loading conditions, for which the exact solution is not available, may be solved with equal ease and accuracy.

#### APPENDIX C

Many instances occur in structural engineering in which continuous loading occurs over only a part of the span. A very close approximation of the deflections due to such a load distribution is obtainable by the use of equation (6). The following example will demonstrate the method of solution. Let it be required to determine the deflection at point B of the beam loaded as shown in figure 4a. The lateral load may be replaced by a number of small concentrated loads of magnitude w dx, where w varies from wo to 2wo, and the deflection at point B

obtained as a sum of the deflections due to the small concentrated loads w dx. When a concentrated load w dx is acting on the beam, let the beam deflection be designated dy. Then from equation (6)

$$dy = \frac{l}{P} (\varphi_{1_C} + \varphi_{2_C}) w dx$$

and 
$$y = \frac{1}{P} \int_{x=0.4l}^{x=0.6l} (\phi_{1c} + \phi_{2c}) w dx$$

$$= \frac{1}{P} \left\{ \int_{x=0.6l}^{x=0.5l} (\phi_{1c} + \phi_{2c}) w dx + \int_{x=0.5l}^{x=0.6l} (\phi_{1c} + \phi_{2c}) w dx \right\}$$

If it were possible to express  $\phi_1$ ,  $\phi_2$ , and w in c c c terms of x the value of y could be accurately determined. This procedure, however, is usually either too difficult or impossible. A very close approximation may be obtained by substituting  $\Delta x$  for dx and replacing the integral by a summation of a finite number of terms.

٧.

At point A

$$u = (1 - \frac{d}{l}) - (1 - \frac{\pi}{l})$$
 (See p. 10.)

 $= (1 - 0.4) - (1 - 0.5)$ 
 $= 0.1$ 
 $v = (1 - \frac{d}{l}) + (1 - \frac{\pi}{l})$  (See p. 10.)

 $= (1 - 0.4) + (1 - 0.5)$ 
 $= -1.1$ 

See table IV for value of  $\phi_{1c}$  and  $\phi_{2c}$ 

$$\phi_{1_{C}} + \phi_{2_{C}} = 0.2309$$
 $\phi_{1_{C}} + \phi_{2_{C}} = 0.2900$ 
 $(\phi_{1_{C}} + \phi_{2_{C}}) = 0.2900$ 
 $\phi_{1_{C}} + \phi_{2_{C}} = 0.2900$ 

At point B

$$u = (1 - 0.5) - (1 - 0.5) = 0$$

$$v = (1 - 0.5) + (1 - 0.5) = 1.0$$

$$\varphi_{1c} = 0.2400$$

$$\varphi_{2c} = 0.0659$$

$$\varphi_{1c} + \varphi_{2c} = 0.3059$$

$$(\varphi_{1c} + \varphi_{3c}) w = 0.3059 (1.5 w_{0}) = 0.4589 w_{0}$$
At point U

$$u = \frac{d}{t} - \frac{x}{t} \qquad (See p. 10.)$$

$$= 0.6 - 0.5 = 0.1$$

$$v = \frac{d}{t} + \frac{x}{t}$$

$$= 0.6 + 0.5 = 1.1$$

$$\varphi_{1c} = 0.2309$$

$$\varphi_{2c} = 0.0591$$

$$\varphi_{1c} + \varphi_{2c} = 0.2900$$

$$(\varphi_{1c} + \varphi_{2c}) w = 0.2900 (2 w_{0}) = 0.5800 w_{0}$$
In this illustrative problem, let  $\Delta x = 0.1t$ 

$$x = 0.5t$$

$$x = 0.4t$$

$$\varphi_{1c} + \varphi_{2c}) w dx = \frac{0.2900 + 0.4589}{2} w_{0} (0.1t)$$

$$= 0.0575 w_{0}t (approx.)$$

$$\varphi_{2c-0.5t}$$

$$\varphi_{1c} + \varphi_{2c}) w dx = \frac{0.4589 + 0.5800}{2} w_{0} (0.1t)$$

$$= 0.0520 w_{0}t (approx.)$$

Therefore the approximate deflection of the beam at point

B is given by

$$y = \frac{l}{P} \left\{ 0.0375 \text{ wol} + 0.0520 \text{ wol} \right\}$$
$$= 0.0895 \frac{\text{wol}^2}{P}$$

If less accuracy is sufficient the trapezoidal loading in figure 4a may be replaced by a single concentrated load of magnitude

$$1.5 \text{ w}_0 (0.2l) = 0.3 \text{ w}_0 l$$

located at the centroid of the trapezoid. The centroid is located at

$$x = 0.4l + 0.556 (0.2l) = 0.5112l = 0.5l (approx.)$$

From equation (8) and table V the deflection at point B is given by

$$y = \frac{l\alpha}{P} (0.3 \text{ wol}) (0.5098) = 0.0918 \frac{\text{wol}^2}{P}$$

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TABLE I

N	Ę 1	tor p	0.06	0.10	0.15	0.50	0,85	0.50	0.55	0.40	0.45	0.50	0,55	0.60	0,65	0.70	0.75	0.20	0.85	0.90	0.95	True p
L	7;	tor T	0.95	0.90	0.86	0.80	0.75	0.70	0,85	0.60	0.55	0.50	0,45	0.40	0.55	0.30	0.85	0.20	0.15	0.20	0.06	for Y
	Values of $\beta_0$ and $\gamma_0$																					
	0		0.1593	0.251.2	0.3860	0.4754	0.5897	0.5978	0.6174	0.851.6	0.650.0	0,6168	0.5905	.0.5527	0.5052	0.4491	0,3866	0.6100	0.2148	0.1598	0.0820	0
ļ	Walnes of β <sub>0</sub> and γ <sub>0</sub>																					
	.8	#8#843#8##5#8#5# 	0.0079 .0163 .0267 .0367 .0459 .0693 .0703 .1070 .1293 .1894 .2951 .2951 .3430 .4457 .6165 .9616	0.0148 .0304 .0478 .0606 .0677 .1111 .1576 .20451 .9451 .9451 .9451 .9451 .9451 .9451 .9451 .9451 .9451 .9451 .9451 .9451 .9451	0.0801 .0428 .0563 .0563 .1294 .1294 .1294 .2367 .2460 .5010 .5010 .5010 .5010 .7664 1.200 1.740 2.710 5.512	0.0946 .0518 .0518 .0514 .1143 .1811 .1935 .2539 .3539 .4301 .5193 .5300 .7704 .909 1.503 2.503 1.503 2.503 7.505	0.0982 .0994 .0994 .1719 .2719 .2719 .3399 .4125 .7901 .0904 1.125 1.099 2.048 4.155 6.070	0.0308 .0847 .1081. .1440 .1910 .948 .3081. .5788 .4581. .5866 .5785 .8940 1.033 1.284 1.284 1.284 2.133 3.000	0.0394 .0683 .1095 .2096 .2096 .2096 .4011 .4091 .6967 .7268 .2097	0.0332 .0700 .11570 .2090 .2032 .3385 .4180 .5092 .4280 1.147 1.428 1.845 3.480 3.480	0.0339 .0701 .1114 .2104 .2710 .2309 .4309 .5189 .6810 .7707 .9453 1.171 1.479 2.519 3.571 5.586	0.0326 .0097 .1093 .1951 .2071 .2077 .4163 .5114 .6361 .7043 1.168 1.471 1.695 2.639 3.591 5.711	0.0312 .0559 .1050 .1491 .1995 .8071 .4085 .4085 .4085 1.139 1.438 1.438 1.438 1.438 1.438 1.438	0.0892 .0819 .0865 .1402 .1402 .9069 .5008 .5008 .5708 .7063 .7063 .7063 .1.094 1.709 1.709 3.309 3.377 5.377 1.45	0.0867 .0866 .0904 .1986 .1785 .2229 .8115 .8556 .4394 .6518 .608 1.098 1.871 1.871 2.806 5.108	0.0638 .0504 .0606 .1140 .1940 .35138 .3275 .4769 .5066 .7254 .9044 1.1463 1.998 9.846 9.846 9.846 9.709	0.0004 .0434 .0090 .0098 .1837 .1717 .2173 .2711 .3350 .4127 .8090 .6867 .7945 .9947 1.994 1.724 2.476 2.476	0.0157 .0355 .0656 .0616 .1090 .1412 .1760 .3409 .4151 .5191 .6484 2.065	0.0138 .0372 .0436 .0634 .1061 .1118 .1711 .2117 .2611 .2117 .2614 .4100 .4064 .8384 1.105 1.305 1.305 1.305	0.0086 .0184 .0894 .0430 .0084 .0731 .0945 .1186 .11768 .1183 .2762 .2762 .2761 .2771 .7514	0.0043 .0093 .0148 .0218 .0259 .0468 .0594 .0769 .1101 .1208 .1708 .2171 .8199 .8769 .8769 .8769	0.05 .10 .15 .20 .25 .20 .40 .45 .80 .85 .90 .85 .90 .85
L				· 	<del></del>	<u> </u>					Yeles	m of A	and 74						·			
111111111111111111111111111111111111111	.06 .10 .30 .30 .30 .40 .56 .60 .90 .90 .90 .90 .90 .90 .90 .90 .90 .9		0.0074 .0143 .0294 .0394 .0394 .0453 .0453 .0453 .0652 .0652 .0762 .0762 .0762 .0962 .0962 .0963	0.0135 .0861 .0861 .0468 .0666 .0666 .0679 .0679 .1193 .1340 .1470 .1540 .1470 .1653 .1790 .1653 .1797 .1653 .1797 .1898 .1997 .2186	0.0188 .0329 .0612 .0672 .0618 .0647 .1074 .1194 .1518 .1518 .1518 .1710 .1801 .1801 .1801 .1801 .1801 .3005 .3148 .3205 .2376 .2376 .2376 .3388	0.0928 .0437 .0632 .080.5 .0984 .1458 .1458 .1458 .1817 .1953 .2048 .2348 .2348 .2348 .2348 .2353 .2360 .2376 .2376	0.0858 .0496 .0717 .0820 .1111 .1800 .1900 .2038 .8677 .2508 .2508 .2708 .2508	0.0883 .0587 .0786 .0965 .1986 .1886 .1886 .2086	0.0886 .0084 .0014 .1140 .1140 .1159 .1800 .2109 .2109 .2589	0.0608 .0676 .0637 .1068 .1870 .1468 .1580 .1891 .1992 .2387 .2587 .2587 .2687	0,0500 .0072 .0093 .1003 .1204 .1204 .1204 .1204 .2009 .2204 .2009	0.0994 .0059 .0059 .1025 .1296 .1419 .1994 .1994 .2157 .2399 .2457 .2399 .2457 .2565 .2565 .2576	0.0891 .0584 .0764 .1165 .1268 .1584 .1585 .1981 .2041	0.0963 .0498 .0498 .0908 .1966 .1400 .1588 .1965 .1892 .1893 .2088 .2557 .8478 .8540 .8540 .8580	0.0339 .0454 .0850 .0838 .0938 .11,896 .151,1618 .1714 .1806 .1953 .1954 .2110 .2110 .2110 .2235 .2331	0.0813 .0404 .0677 .0733 .0676 .1128 .1128 .1284 .1486 .1591 .1685 .1797 .1816 .1914 .2013 .2013 .2013 .2013 .2014 .2013 .2014 .2013 .2014 .2013 .2014 .2013	0.0083 .0645 .0646 .0690 .0690 .0691 .1054 .1217	0.0149 .0985 .0404 .061.8 .0708 .0708 .0969 .0969 .1061 .1165 .1204 .1265 .1269 .1469 .1469 .1468 .1548 .1668	0.011A .0817 .0803 .0393 .0456 .0555 .0707 .0758 .0769 .0515 .0976 .0116 .1040 .1054 .1040 .1145 .1145 .1340	0.0077 .0047 .0064 .0064 .0064 .0407 .0408 .0408 .0508	0.0036 .0073 .0103 .0133 .0183 .0296 .0299 .0285 .0297 .0390 .0390 .0390 .0390 .0390 .0390 .0390 .0390 .0390	0.05 .10 .10 .25 .25 .25 .25 .25 .25 .25 .25 .25 .25

TABLE II

a z	0.05	0.10	0.15	0.20	0.25	0.30	0.35	0.40	0.45	0.50		
Values of 80												
0	0.0205	0.0403	0.0591	0.0781	0.0916	0.1405	0.1148	0.1824	0.1270	0.128		
Values of 60												
.10 .15	0.0011	0.0021 .0045 .0071	0.0031 .0085 .0104	0.0040 .0085 .0134	0.0049 .0102 .0161	0.0055 .0116 .0184	.0128 0.0060	0.0065 .0136 .0216	0.0067 .0141 .0284	0.008 .014 .028		
.20 .25 .30	.0051 .0068 .0087	.0100 .0134 .0172	.0147 .0196 .0253	.0191 .025 <u>4</u> .0326	.0338 .0305 .0392	.0261 .0348 .0447	.0387	.0306 .0408 .0525	.0317 .0423 .0544	.032 .048 .056		
.35 .40 .45	.0110 .0136 .0167	.0216 .0268 .0328 .0401	.0317 .0393 .0481 .0588	.0410 .0508 .0622 .0760	.0492 .0609 .0748 .0913	.0582 .0697 .0855 .1044	.0618 .0766 .0940 .1148	.0859	.0684 .0848 .1040	.069 .085 .105		
.55	.0249 .0305 .0376	.0490 .0601 .0744	.0719 .0882 .1091	.0929	.1116 .1369 .1696	.1277 .1587 .1939	.1404	.1498 .1838 .2277	.1555 .1909 .2363	.157		
.70 .75 .80	.0473 .0607 .0812	.0935 .1199 .1599	.1371 .1761 .2350	.1773 .2277 .3038	.2132 .2740 .3854	.3438 .3138 .4178	.2682 .3446 .4601	.2862 .3677 .4908	.2971 .3818 .5095	.300 .386 .516		
88. 90. 86.	.1147 .1830 .3840	.2265 .3595 .7586	.3328 .5279 1.114 &	.4305 .8834 1.443	.5200 .8217 1.735	.5960 .9398 1.985	.6519 1.035 2.186	.6958 1.104 2.333	.7222 1.147 2.422	.731 1.161 2.453		
	L	L	L	<u></u> ,	Values (	of at	<u> </u>	<u> </u>	<u>l</u>	<u>l</u>		
0.05	0.0010	0.0019	0.0028	0.0038		0.0050	0.0055	0.0058	0.0060	0.008		
.10	.0019	.0037	.0054	.0089	.0083	.0095	.0105	.0111 .0160 .0204	.0115	.016		
.20	.0034	.0068	.0099	.0127	.0152	.0175	.0230	.0245	.0254	.021		
.30 .35	.0047	.0094	.0137	.0176	.0211	.0241	.0265	.0283	.0393	.033		
.40	.0059	.0118	.0170	.0218	.0262	.0298	.0328	.0349	.0363	.036		
.45	.0069	.0135	.0198	.0255	.0305	.0348	.0383	.0407	.0422	.042		
.55 .60	.0073	.0144	.0211	.0272	.0326	.0371	.0408	.0434	.0450	.048		
.65	.0081	.0160	.0234	.0302	.0361	.0412	.0452	.0482	.0499	.080		
.70 .75	.0085	.0167	.0245	.0315	.0378	.0430	.0492	.0583	.0521	.088		
.80	.0092	.0181	.0265	.0341	.0408	.0465	.0510	.0543	.0563	.057		
	CMAN	.0187			.0435	.0495	.0544	.0579	.0600	.080		
.85	.0098	.0193	.0282	.0363								
.85 .90	.0098	.0198	.0280	.0374	.0447	.0510	.0559	.0595	.0617	.062		
.85 .90 .95 .00 1.00	.0098 .0101 .0104 .0115	.0198 .0204 .0227	.0290	.0374 .0384 .0427	.0447 .0459 .0510	.0510 .0523 .0581	.0575	.0612	.0617 .0634 .0703	.064		
.85 .90	.0098 .0101 .0104	.0198 .0204 .0227 .0245	.0290	.0374	.0447	.0510	.0575	.0612	.0617	.064 .064 .071 .076		

TABLE III

V	0.06	0.10	0.15	0.30	0.85	0.30	0.35	0.40	0.45	0.60	0.55	0.60	0.65	0.10	0.75	0.60	0.65	0.90	0.95	
Values of co																				
0	0.0096	0.0189	0.0379	0.0663	0.0488	0.0504	0.0559	0.0801	0,0699	0,0545	0,0841	0.0625	0.0590	0.0661	0.0479	0.0401	0.0513	0.0214	0.000	0
	Talmes of co																			
0.05	0.0005	0.0011	0.0018	0.0018	0.0023	0.0027	0.0050	0.0058	0.0054	0.0055	0.0055	0.0039	0.0031	0,0099	0.0095	0.0019	0.0018	0.0010	0.0005	0.08
.10	.0011	.0021	.0051	.0061	.0049	.0086	.0068	.0087 .0108	.0070	.0071	.0071	.0069	.0088	,0059 3900.	.0063	.0045	.0055	.0023	.0011	] .10 31.
.90	.0024	.0048	.0070	.0001	0110	.0198	.0140 .0187	0151	.0157	.0150 .0214	.0010	01.55	,0147 ,0195	0134	.0119	.0099	.0077	.0053	.0036	.90
.95	.0058	.0064	.0094	.02.67	.03.89	.0218	.0911	0200	,0270	.0976	.0074	.0956	.0268	.0230	.0905	.03.60	.0138	.0090	.0046	.50
.55	.0065	.0105	.01.68 .01.60	.0098	8820.   8820.	.0274	.0306 .0376	.0385	.0540	.0348	.0345 0427	.0334	.0315 .0390	.0988	.0254	.081.9 .0953	.0386	.0113	.0058	.40
48	.0000	.0198	.0869	.0301	.0384	.0418	,046g	.0505	.0517	.0896	.0695	.0507	.0478	0457	.0384	.0831	.0349	.0171	.0087	.46
.55	.0146	.0987	.0548	.0468	.0545	.0536	.0891	.0741	.0972	.0795	.0790	.0755	.0711	.0849	0571	.0177	.0389	.0359	.01.98	.65
.83	.0165	.0208	.0555	.0691	.0686	.0965	.1064	.1130	.1177	.1198	.1106	.11,48	,1079	.0984	.0863	.OTEO	0559	.0361	.0004	,66
. TO	.0951 .0998	.0457	.0872	.0870	1997	.1205 -1548	.1598 .1709	.14#1 .1898	.1461	.1504 .1952	.1401 .1914	.1441	.1555 .1738	.1865 .1863	.1085 .1388	.0904	.0896	.0479	.0270	.70
.80 .85	.0399 3630,	.0767	.1156 .1644	.1409	.1906	.9068 GEGE.	.8982 3976	.0440 .3464	.2630	.3677	.9651 .8618	.3453 .3489	.5315 .3276	.EL06	.1846 .2609	.1637 .2173	.1190 .1681	.0812	.0611	.80 .85
.90	.090B	.1763 .3780	.361.9 .5553	.5590 .7197	.4085 8653	.4677	.5155 1.001	.5608 1.165	.5726 1.211	.8905 1.297	1.913	1.158	.5192 1.095	.4780	.4151 .8704	.3439	.2659	.1811 .5810	.0917	.90 .95
1.00		.0.0				•		•	•	•						•		•		1.00
		1	<del></del>	1	<del> </del>	I			Velm		·	T=====	la				Ta		la ese	1
0.05 10.	0.0005	0.0009	0.0015	0.0017	0.0021	.0046	0.0097 .0051	0.00 <b>3</b> 8	0.0030	0.0051. 9500.	0.0001	0.0050	0.0098 .0064	0.0026	0.0025 ,0044	0.0019	.0099	.0019	.0010	110
, 15 . 20	.0012	.0095	.0036	.0047	.0057	.0065	.0072	.0078	.0082	.0064	.0084	.0081	.0076	.0071	.0063 .0081	.0055	.0041	.0098	.001.6	1.25 25.
.25	.0019	.0058	.0066	0078 0083	.0067	.0100	.0118	.01.50 .01.58	.01.45	.0107 .0139 .0148	.0128	.0125 .0144	.0119	80.00. 88.70.	.0086	.0081	.0054	.0044	.0088	.#5 .50
. 35	.0034	.0048	.0071	.0095	0112	.0120	.0144	.0155	.0162	.0165	.07.66	0162	.0154	.0141	.01.86	.0108	.0089	.0000	,0089	. 35
.43	.0097	.0067	.0076	.0102	.01.83	.01,43	.0158 .0178	.0170 .0185	.0179 .0194	0199	.0185	.0194	.0195	.0170	.0158	.0127	.0098		.0035	.49
.50 .55	.0003	2800. 8800.	.0090	0110	.0144	.0158	.0194	.0130	.0909	0404	.020.4 .0228	.0908	.0198	.0189	.0168	.0136	.0108	.0078	.0057	.50 .55
.60	.0035	.0010	0100	.0134	.0168	.01.86 .01.98	.0907	.0225	.0935	.0253	.0253	.0235	.0924	.0908	.0193	.0154	.0120	.0087	.0014	.65
.70	.0030	.0076	.0111	.0146	.0177	.0204	.0997	.0334 .0345 .0355	0957 0958	.0864	0054	.0239	.0945	.0827	.0901	.01.00	01.50	.0093	.0046	.70 .75
.75 .80	.0040	.0079	.01.93	.01.07	.0180	.020.2 0227	.0945	.0964	.0976	,0986	.0985	.0379	,0965	.0945	.0017	.0188	.0143	.0000	.0050	.80
.85	.0044	0080	.01.85	.00.63	.0197	.0634	.0255	.0961	.0297	.0304	.0604	.0988	.0975	.0953	.0825 .0838	.0190	.07,40 .07,64	.00.00	.0052	.85
1.00	.0045	.0080	.0139	.0170	.0908	.0940	.0867	.0989	.0804	.051.8	.051.5	.0308	.0202	.0250	.0239	.0909	.0158	.0110	.0086	1.00
1.50	,0053		01.6	03.95	,0000	.0273	.0504	.0598	.0347	.0555	.0557	.0380	.0854	.0308 .0384	.0974	CSSSE	.01.82		,0064	1.85
1.75	.0066 .0066	0110	.0151	OHAT.	.0864	.0304	.0587 .0346	.0554	.0575 .0395	.0594	.0588	.0378	.0361	.0388	.0898	.0988	.0811	.0146	.0010	
8.00	.0080	,0120	.0277	.0131	.0990	.0524	.0368	.0992	.061.5	.04.98	.0420	.0430	.0403	.0374	.0334	.0265	.0292	.01.54	.0076	2.00

TOTA IV

O BOOL W	0	0.1	0.8	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0		
					<u> </u>			<del></del>		<del> </del>	<del></del>	U end v	
	9.0	1.9	1.8	1.7	1.6	1.5	1,4	1.8	1,8	1.1	1,0	<u> </u>	
Wellson of the and -the													
0	0.1096	0.1080	0.0850	0.0558	0,0966	-0.0059	-0.0654	-0.0608	-0.0798	-0.0919	-0.0969	0	
	Falmes of T <sub>20</sub> andT <sub>20</sub>												
0.06	-0.8408	-0.8419	-0.8492	-0.8456	-0.8455	-0,8468	-0.8485	-0.8408	-0.6509	-0.8834	-0.8517	0.05	
.1 .8	2980 0595	5987 0611	5999 0689	3387 0726	~.3571 ~.0801	3405 0879	3489 0963	3457 1018	3489 1087	-,5805 -,1098	3507 1108	.1	
	.1112	.1059	.0968	,0783	.0583	.0374	.0178	0002	0157	0688	C267	1 .4	
.6	.9400	.2309	.2060	.1693	.1205	.0790	.0178 .0395 0961	0079	0802	0591	0650	.6	
.8 .9	.51.66 1.035	.4045 .9845	.4305 .8406	.5365 .6428	. 3226 . 3856	.0973 .1034	0961	0560 4970	2223 6260	3774 7517	9964 7955	.8 .9	
.95	2.050	1.965	2.485	1.243	7010	1006	4878	-1.081	-L442	-1.713	-1.808	.96	
.975	4.071	5.6T4	. 5,508	2,430	1.395	.1069	-1.110	-8.205	-5.075	-5.053	-5.625	.975	
1.0 1.025	-4.054	-5.939	-5.247	-2.335	-1.177	.1099	1,897	8,561	3.497	4.083	4,266	1.0	
1.05	-1.996	-1.897	-1.801	-1.156	5471	1098	.7898	1.367	1.948	2.148	3.354	1.05	
1.1	9861	9309	7794	NOA	~.2530	.1114	,4595	.9741	1.098	1,166	1.244	1.1	
1.8	4500	4480	3658	- 2380 - 0845	0752	.1140	.5048	4797	.6198	.71.06	.7418	1.3	
1.4	~.9073 ~.1146	1945 1066	3636 0601	0845 0518	.0085 .0358	.1174 .1187	.2088	.3358	.4210 -5592	.4778	.4971 .4303	1.4	
1.8	0544	0598	0424	0050	.0486	.1186	.1945	.2682	.5398	.2707	.5000	1.8	
8.0	0311	0889	0159	,0058 ,0297	.0571	.1168	.1951 .1779	.8550	.33.36	.2020	.3588	8.0	
1.25	0	0007	.0049	.0227	.0593.	.1188	.1779	.9441	.3084	.5480	.5551	1.15	
2.5 2.75	.0952 .0530.	.0226	.0554	.0305	.0608	.1052 .0867	.1660 .1664	.2284	.9961	.5388	.3040 2005	2.75	
3.0	.0840	.0700	.0157	.0307	.0377	.0761	1396	2199	.8979	.54.98	.oraa	3.0	
5.25	.1938	.1048	.0626	.0041	.01.26	.0427	.1197	.2100 .2000	. 5045	.2756 .4266 .8676	.4051	5.86	
3.8 3.75	.1980 .4065	.1851. .3340	.0677	.0051 0545	~.0897 ~.8088	0336 9261	.0564 1050	.1953 .1253	. 3850 . 3858	.4908	.4655 .8647	3.10	
5.9		.8878	.5439	3406	-,6980	8813	8960	0643	.5708	1.077	1,350	3.9	
3.95	1.016	1.054	.6591,	5668	-1.518	-1.888	-1,491	5799	.8856	1.902	9.200	8.95	
4.00			-	-	Values	of the and	-	-		<u> </u>		4.00	
0.1	-0.6654	-0.5690	-0.4858	-0.4133	-0.3811	of the end	-0.2556	-0.2230	-0.2005	-0.1267	-0.1880	0.1	
.8	401.B	~. <b>3078</b>	2961	1887	-,0959	0461	-,0060	-0.3930	.0458	.0898	.0541		
	3098 2613	2155	- 1384 - 0896	0678 0686	0100 5556	.0795	.0761	.1054	.1861	.1385 .1758	-1486	, š	
: i	8304	1566 1562	-,0508	.0065	.0386	.1044	.1398	.1007	1888	.1766	.1797 .2006	3.5	
.6 .6	9097	- 1169 - 1006	0388	.0944	.0776	.1,908	.1551 .1657	.1816	.1993	.2101	.2137	.6	
-7	1982	1008	0947	.0494	.0905	.1334	.1657	.1909	.8086	.91.80	. 2004	.7	
.ė. .ė.	1792 1686	0888 0779	01.29 0035	.0481	.1000 .1075	1410	.1758	.1979 .2051	.21.60 .21.66	2361	. 2506 . 2530	.8   .9	
1.0 I	-, 159?	0779 0694	.0044	.0846	.1136	.1590	.1938 .1918	.9071 .9138	. 2854	.2997 .2550	.2363	1.0	
1,45	1498	→.0F30	.0199	.077Б	.3245	.1622	.1918	21.39	. 2995	.2586	.2116	j 1.95	
1.5	1301	-,0411 -,0320	.0996	.0063 3860	.1518	1589	.1965	.21.78 .2904	.2587	.2415	.2445	1.5	
1.75 8.0	1205 1195	0247	.0427	.0978	.1409	.1799 .1760	2019	.8991	.2052	.2448	.3462 .3474	1.75 2.0	
#.V				100							1,077		

ηī	A	R	7	T.	v

		TAB	V نظر		
$\alpha \frac{x}{l}$	0.1	0.2	0.3	0.4	0.5
		Values	of ζ <sub>O</sub>		
0	0.0266	0.0842	0.1451	0.1895	0.2056
		Values	of \$c		
0.2	0.0318	0,1025	0.1788	0.2355	0.2563
. 4	.0401	.1324	.2349	.3124	.3411
.6	.0566	.1914	.3461	.4654	<b>.</b> 50 98
.8	.1053	.3671	. 6782	.9235	1.016
. 9	.2023	.7179	1.342	1.841	2.030
.95	.3963	1.420	2.671	3.676	4.059
.975	.7816	2.814	5.313	7.328	8.096
1.0	<b>∞</b>	<b>~</b>	<b>&amp;</b>	8	œ
1.025	7662	-2.786	<b>~5.297</b>	-7.336	-8.115
1.05	3779	-1.381	-2.636	-3.657	-4.049
1.1	1842	6814	-1.309	-1.825	-2.023
1.2	0872	3305	6456	9081	-1.010
1.4	0383	1542	3129	4494	5032
1.6	0216	0943	2009	2961	3342
1.8	0128	0633	1439	2190	2497
2.0	0071	0437	1086	1724	1989
2.25	0018	0262	0787	1344	1583
2.5	.0025	0125	0567	1082	1311
2.75	.0068	.0005	0379	0883	1117
3.0	.0117	.0148	0192	0717	0970
3.25	.0189	.0342	.0034	0556	<b></b> 0856
3.5	.0315	.0679	.0399	0357	0764
3.75	.0674	.1623	.1364	•0060	0689
3.9	.1725	.4379	.4131	.1143	- 0649
3.95	.3491	1.466	1.023	.2917	0638
4.0	<b>©</b>	<b>0</b> 2	- ∞	<b>CO</b>	0625
		Values	of \$t		
0.1	0.0248	0.0774	0.1330	0.1730	0.1874
.2	.0232	.0720	.1225	.1588	.1719
.2 .3	.0218	.0662	.1137	.1468	.1587
.4	•0206	.0630	.1060	1366	.1475
•5	.0196	.0595	.0996	.1289	.1379
• 6	.0187	.0563	.0938	.1201	.1295
•7	.0179	.0535	.0887	.1133	.1220
.8	.0171	.0510	.0842	.1072	.1154
•9	.0165	.0487	.0802	.1018	.1094
1.0	.0159	.0467	.0764	.0969	1040
1.25	.0146	.0423	.0686	.0865	.0926
1.5	.0135	.0387	.0623	.0781	.0837
1.75	.0126	.0357	.0570	.0712	.0761
2.0	.0119	.0333	.0527	.0656	.0700
	<del></del>		·		

T.	Δ	R	т.	Τ.	₹	ΙI	
	a	·	-	-	1		

TABLE VI											
$\alpha \frac{\mathbf{x}}{l}$	0.1	0.2	0.3	0.4	0.5						
		Value	s of αζ <sub>c</sub>								
0.4 6.8 9.9 1.0 1.0 1.2 1.6 8.0 1.0 1.2 1.6 8.0 1.0 1.2 1.6 8.0 1.0 1.0 1.0 1.0 1.0 1.0 1.0 1	0.0064 .0160 .0340 .0843 .1821 .3765 .7627 .854 3968 2026 1047 0537 0345 0230 0141 0040 .0062 .0187 .0352 .0614 .103 .2526 .6727 1.379	0.0205 .0530 .1148 .2937 .6461 1.349 2.744 -2.856 -1.451 7495 3966 2159 1509 1509 1509 1509 0312 .0013 .0444 .1112 .2377 .6086 1.708 5.792	0.0358, .0940 .2077 .5426 1.208 2.537 5.180 -5.430 -2.767 -1.441 7747 4381 2589 2172 1417 1043 0577 .0111 .1395 .5114 1.611 4.042	0.0471 .1250 .2792 .7388 1.657 3.493 7.145 -7.520 -3.840 -2.008 -1.090 6292 4738 3942 3944 3024 2705 2430 2151 1850 2458 1.152	0.0513 .1364 .3059 .8128 1.827 3.857 7.893 -8.318 -4.251 -2.226 -1.212 7045 5347 4495 3978 3978 3278 -						
4.0	50	<b>∞</b>	œ	&	2500						
		<del></del>	s of alt								
0.1 .3.4 .5.6 .7.8 .9.0 .5.5 .7.5 1.5.5 2.0	0.0025 .0046 .0065 .0082 .0098 .0112 .0125 .0137 .0148 .0159 .0182 .0203 .0220	0.0077 .0144 .0199 .0252 .0297 .0338 .0375 .0408 .0439 .0467 .0529 .0581 .0666	0.0133 .0245 .0341 .0424 .0498 .0563 .0621 .0674 .0722 .0764 .0857 .0934 .0997	0.0173 .0318 .0440 .0546 .0644 .0721 .0793 .0858 .0916 .0969 .1081 .1171 .1246	0.0187 .0344 .0476 .0590 .0689 .0777 .0854 .0923 .0985 .1040 .1157 .1256 .1332						

TABLE VII										
a \\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	0.1	8•0	0.3	0.4	0.5					
		Values o	f K <sub>lo</sub>							
0	37.59	11.88	6,892	5.277	4.863					
		Values o	f K <sub>lc</sub>							
0 .4 .6 8 .9 5 5 5 5 5 1 .0 .2 .4 .6 8 .0 .2 5 .7 5 5 1 .1 .1 .1 .2 .2 .2 .3 .3 .3 .3 .3 .3 .3	31.48 24.92 17.66 9.494 4.943 2.523 1.27 0.305 -2.646 -25.429 -11.46 -26.36 -78.25 -141.4 -568.2 -141.4	9.756 7.553 5.225 2.724 1.393 .704 .355 0.359 -1.468 -2.026 -15.80 -22.88 -38.17 -80.0 212.8	5.593 4.257 2.889 1.474 .745 .374 .188 0.189764 -1.549 -3.196 -4.979 -6.951 -9.210 -17.64 -26.37 -51.95 292.4	4.246 3.201 2.149 1.083 .543 .272 .136 0136 273 548 -1.101 -2.225 -3.377 -4.566 -5.800 -7.442 -9.240 -11.32 -13.95 -17.99 -28.00 165.6	3.902 2.932 1.962 9.84 4.493 2.46 1.24 0.123 494 990 -1.987 -2.992 -4.005 -6.319 -7.627 -6.319 -7.627 -10.30 -11.68 -13.08 -14.52 -15.41					
3.95					-15.67					
4.0					-16.00					
	<del>~ · · · · · · · · · · · · · · · </del>	Values o	r K <sub>lt</sub>							
0.1 .23.4 .56.7 .89.05.55.0 1.57.00	40.32 43.20 45.81 48.57 51.02 53.53 55.99 58.34 60.72 68.59 68.59 79.68 79.68	12.92 13.88 15.11 15.87 16.81 17.76 18.69 19.60 20.52 21.42 23.64 25.81 28.03	7.519 8.163 8.795 9.434 10.04 10.66 11.27 11.88 12.47 13.09 14.58 16.05 17.54 18.97	5.780 6.297 6.812 7.321 7.758 8.326 8.826 9.328 9.823 10.32 11.56 12.80 14.05 15.24	5.336 5.817 6.301 6.780 7.252 7.722 8.197 8.666 9.141 9.615 10.80 11.95 13.14 14.29					

m	A'	12	٧.	W.	v	Ŧ	T	٣	

TABLE VIII											
$\alpha \frac{x}{l}$	0.1	0.2	0.3	0.4	0.5						
		Values o	f K <sub>sc</sub>								
0.2 .4 .6 .8 .9 .95 .975 1.0 1.025 1.05 1.1	156.3 62.50 29.41 11.86 5.491 2.656 1:312 0 -1.273 -2.520 -4.936 -9.551 -18.62	48.78 18.87 8.711 3.405 1.548 .741 .364 0350689 -1.334 -2.521 -4.632	27.93 10.64 4.815 1.843 .828 .394 .193 0184 361 694 -1.291 -2.283	21.23 8.000 3.582 1.354 .604 .286 .140 0 133 260 498 918 -1.589	19.49 7.331 3.269 1,230 .547 .259 .127 0120235449825 -1.419						
1.6 6 8 0 2 5 5 5 5 5 5 5 5 5 5 5 7 9 9 5 5 7 9 9 5 7 9 9 9 9	-28.99 -43.48 -70.92 -250.0 161.3	-6.627 -2.780 -11.44 -16.95 -32.05 769.2 	-3.111 -3.862 -4.604 -5.643 -7.057 -9.588 -17.33 90.00	-2.111 -2.537 -2.900 -3.307 -3.697 -4.115 -4.649 -5.534 -8.000 44.25	-1.870 -2.225 -2.514 -2.808 -3.051 -3.256 -3.435 -3.593 -3.738 -3.951 -3.968 -4.000						
	,	Values o	<del></del>	<del></del>							
0.1 23.4 5.6 7.8 9.0 1.25 1.75 2.0	400.0 217.4 153.8 122.0 102.0 89.29 80.00 72.99 67.57 62.89 54.94 49.26 45.45 42.02	129.9 69.44 50.25 39.68 33.67 29.59 26.67 24.51 22.78 21.41 18.90 17.21 16.03 15.01	75.19 40.82 29.33 23.58 20.08 17.76 16.10 14.84 13.85 13.09 11.67 10.71	57.80 31.45 22.73 18.31 15.53 13.87 12.61 11.65 10.92 10.32 9.251 8.540 8.026 7.622	53.48 29.07 21.01 16.95 14.51 12.87 11.71 10.83 10.15 9.615 8.643 7.962 7.508 7.143						

Figure 4a.

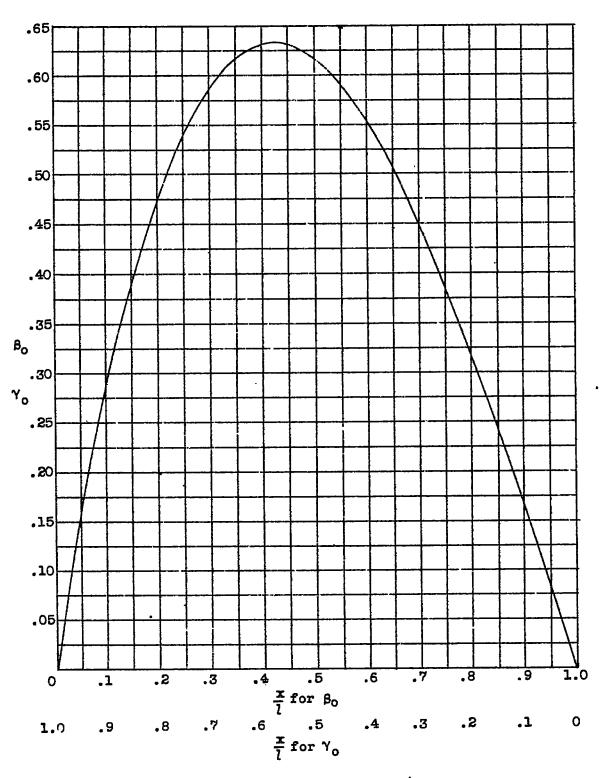


Figure 5.- Plot of  $\beta_0$  and  $Y_0$  against  $\frac{x}{l}$  .

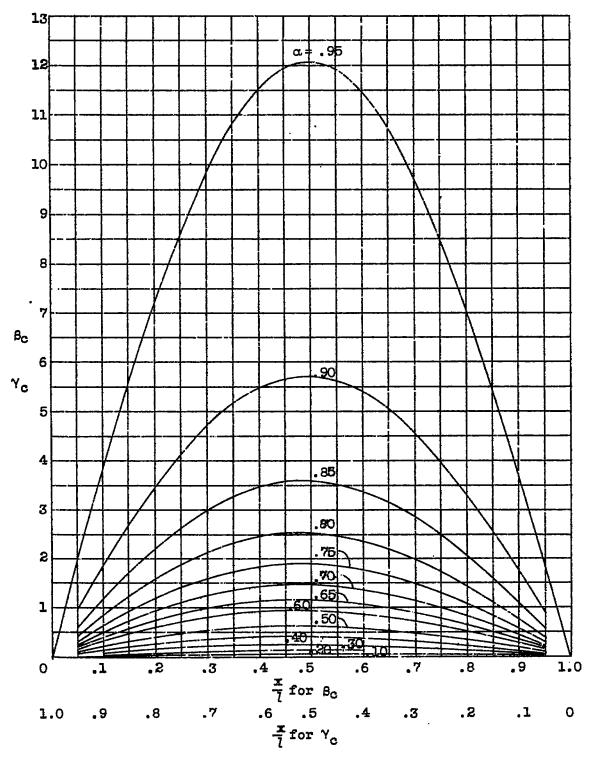


Figure 6.- Plot of  $\theta_{C}$  and  $Y_{C}$  against  $\frac{x}{l}$ .

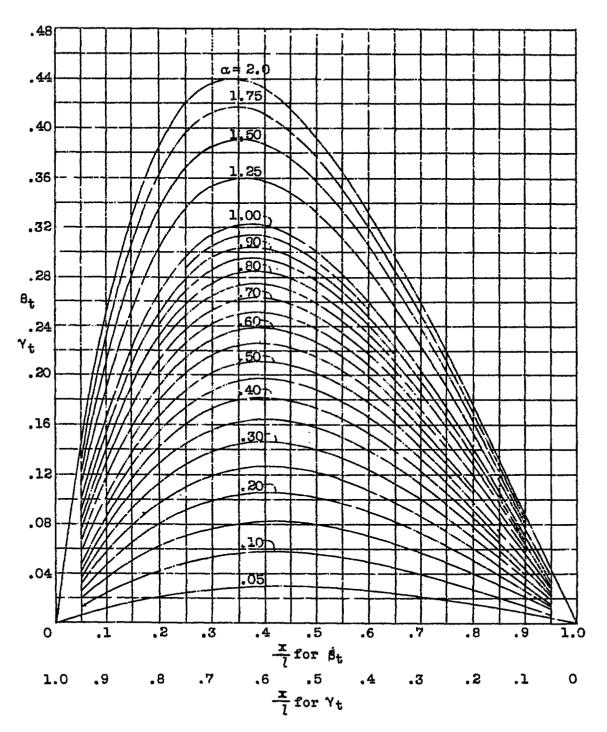
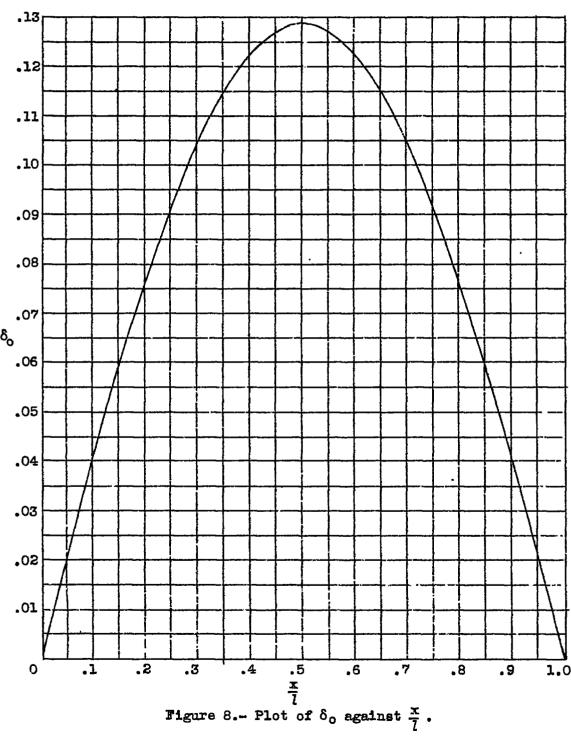


Figure 7.- Plot of St and Yt against  $\frac{x}{l}$ .



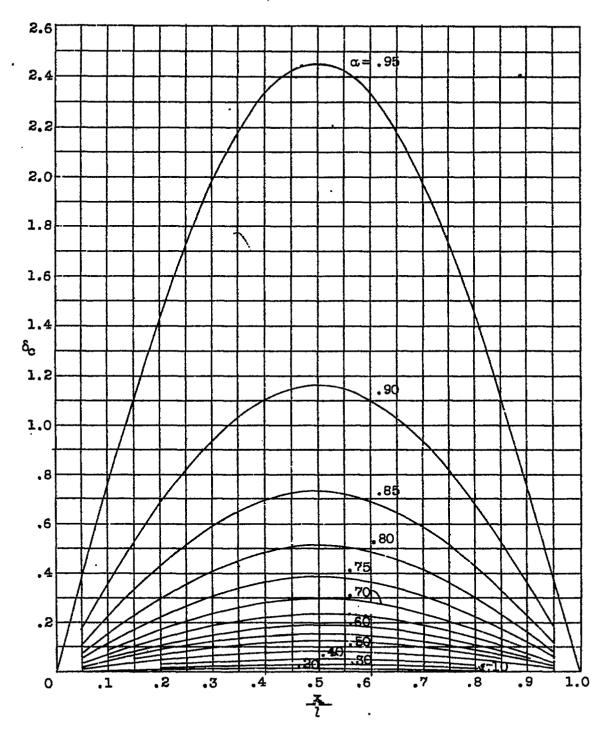


Figure 9.- Plot of  $\delta_0$  against  $\frac{x}{l}$ .

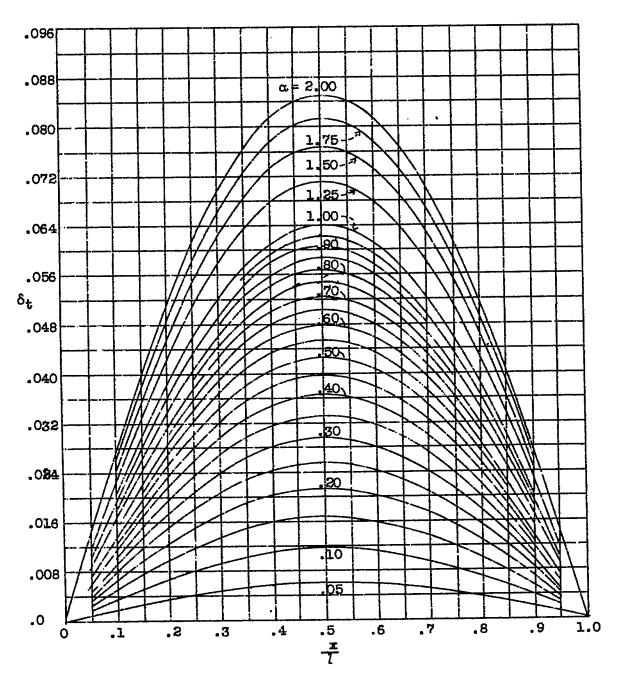


Figure 10.- Plot of  $\delta_t$  against  $\frac{x}{l}$ .

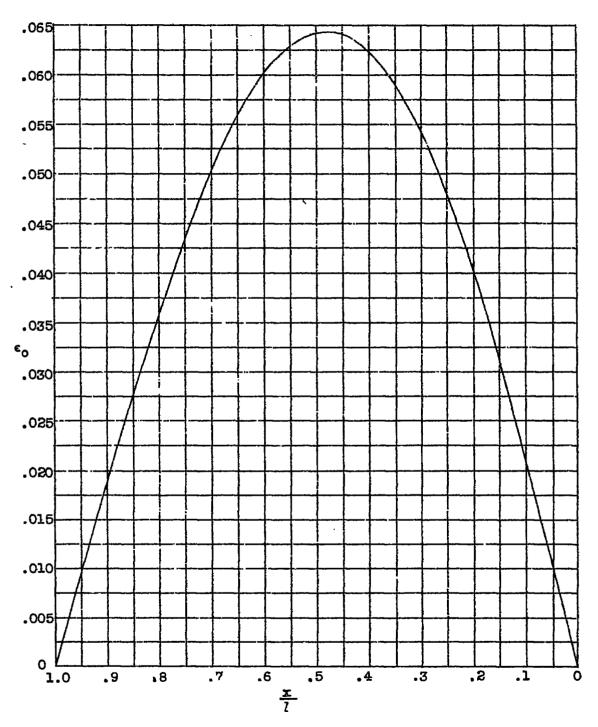


Figure 11.- Plot of  $\epsilon_0$  against  $\frac{x}{l}$ 

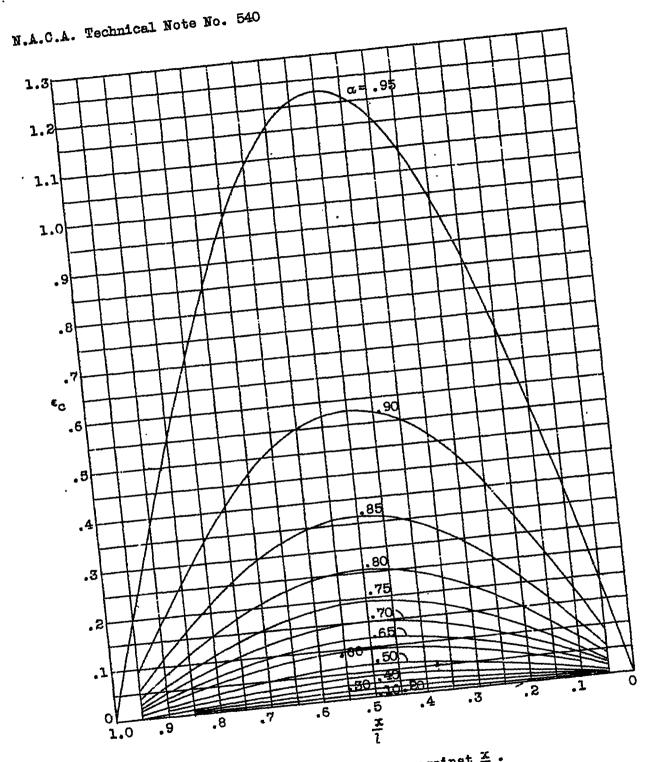


Figure 12.- Plot of &c against 7.

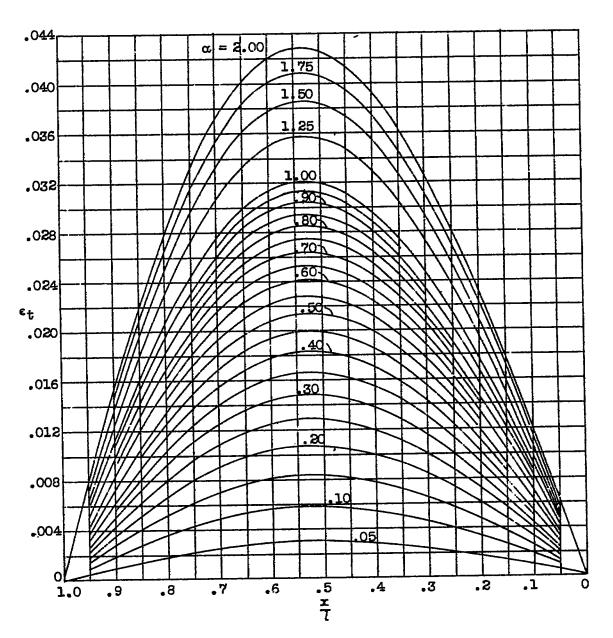


Figure 13.- Plot of  $\epsilon_t$  against  $\frac{x}{l}$ .

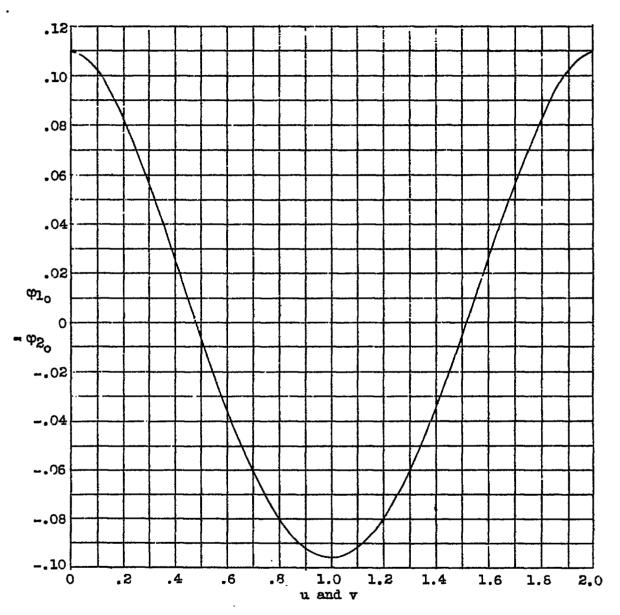


Figure 14.- Plot of  $\phi_{1_0}$  and  $-\phi_{2_0}$  against u and v.

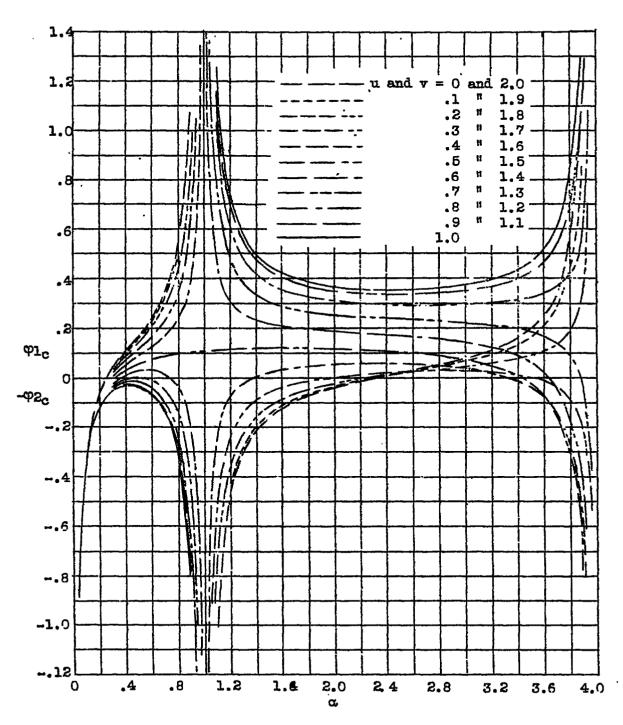


Figure 15.- Plot of  $\phi 1_{\mathbf{C}}$  and  $-\phi_{\mathbf{C}_{\mathbf{C}}}$  against  $\alpha.$ 

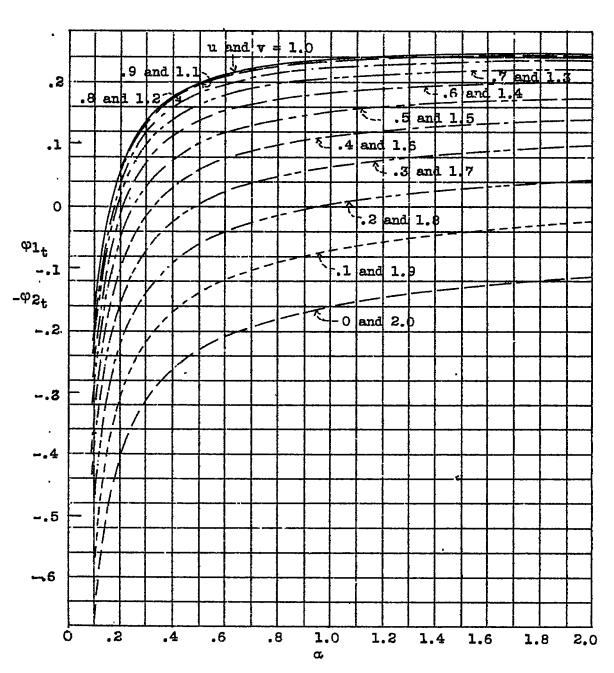


Figure 16.- Plot of  $\phi_{\mbox{\scriptsize l}_{\mbox{\scriptsize t}}}$  and  $\mbox{\scriptsize -}\phi_{\mbox{\scriptsize 2}_{\mbox{\scriptsize t}}}$  against  $\alpha.$ 

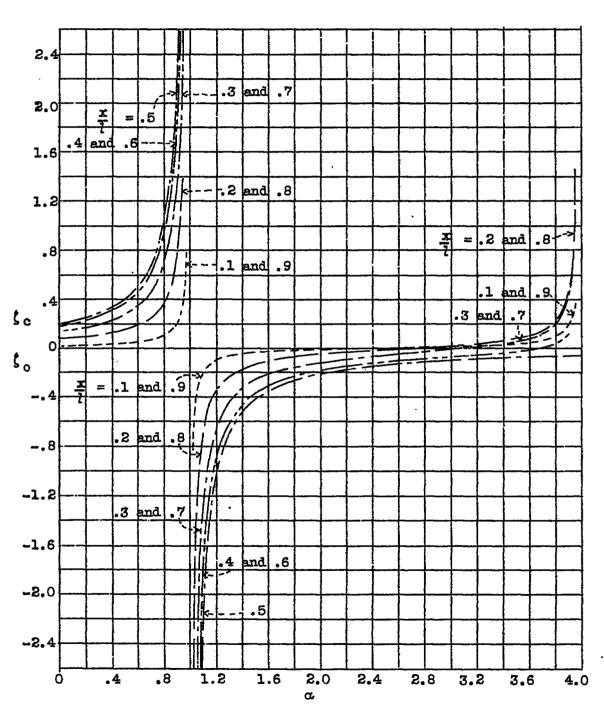


Figure 17.- Plot of  $\zeta_{\rm C}$  and  $\zeta_{\rm O}$  against  $\alpha_{\rm c}$ .

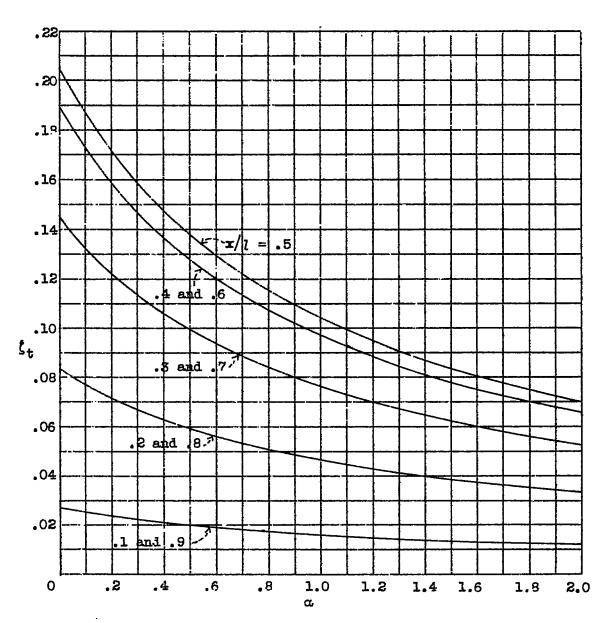


Figure 18.- Plot of  $\zeta_{\rm t}$  against  $\alpha.$ 

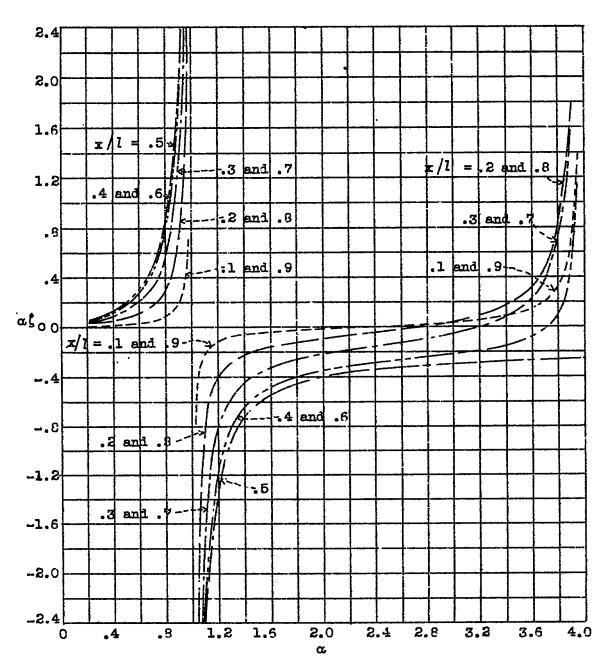


Figure 19.- Plot of a cagainst a.

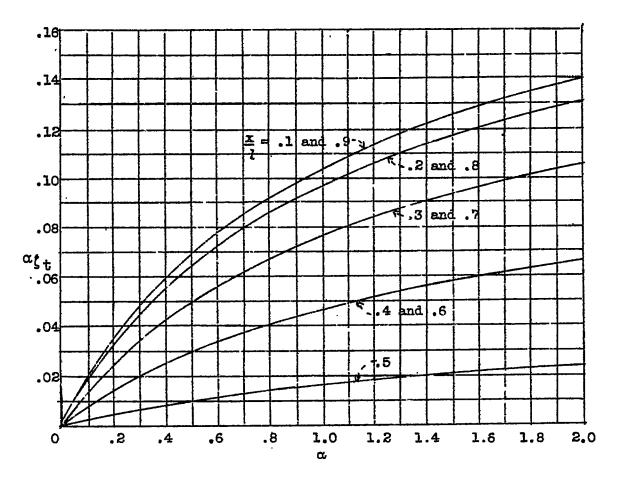


Figure 20.- Plot of  $\alpha_{it}$  against  $\alpha$ .

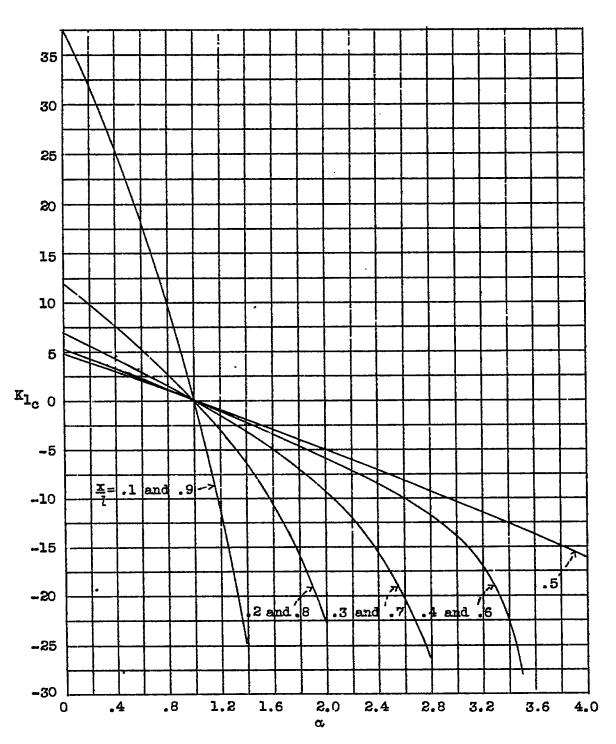


Figure 21.- Plot of  $K_{\mbox{lc}}$  against  $\alpha$ .

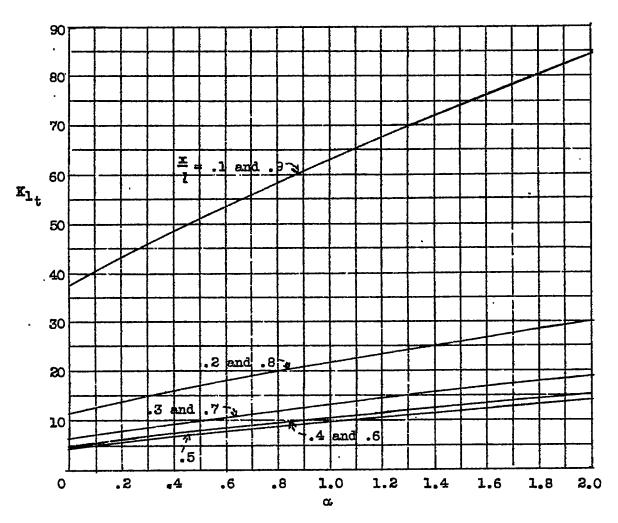


Figure 22.- Plot of  $K_{l_{t}}$  against  $\alpha$ .

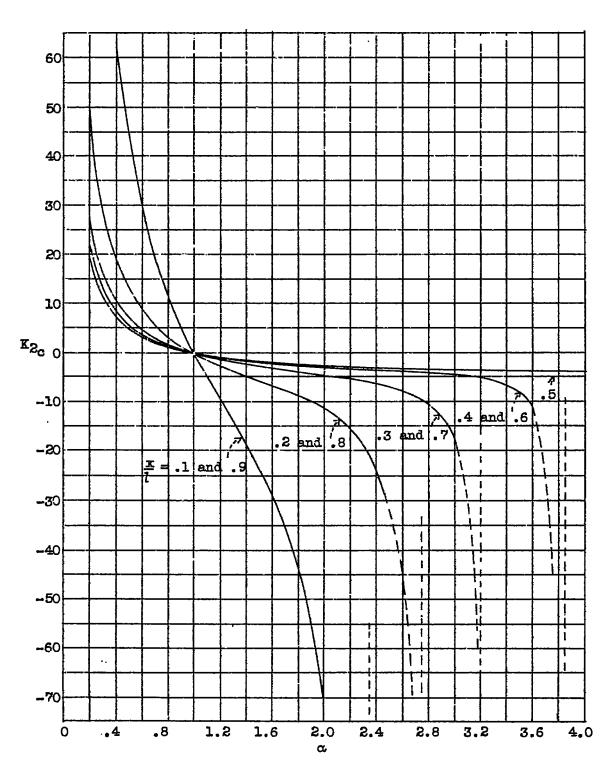


Figure 23.- Plot of  $K_{2_{_{\mathbf{C}}}}$  against  $\alpha$ .

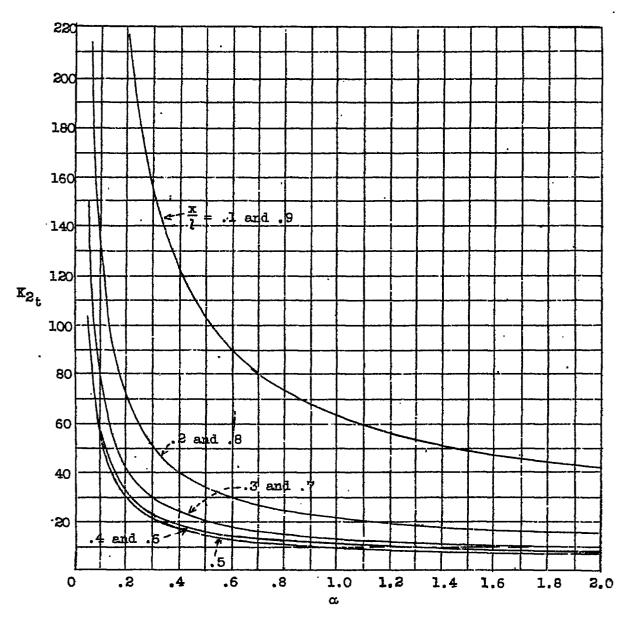


Figure 24.- Plot of  $K_{2t}$  against  $\alpha$ .